College of Maritime Engineering, and Biological, Oceanical and Natural Resource Sciences

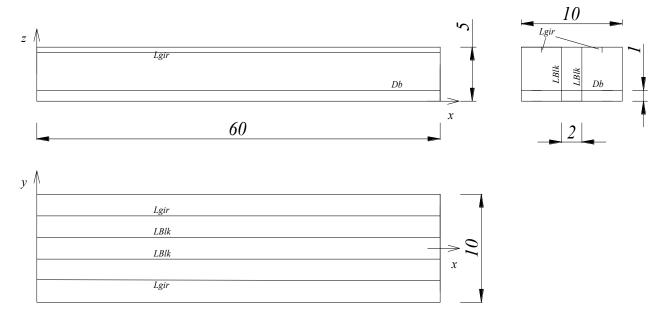
Ship Structures II

Second evaluation

August 27th, 2018

Student:

1.- Calculate the possibility of buckling of the deck plates in a transversely framed steel box barge with the following dimensions, L: 60, B: 10, and, D: 5 m, which transports 1300 tons of load plus consumables, and has a light weight of 200 t. In a simplified manner, you may consider the weight as uniformly distributed. The ship is sailing in sinusoidal waves of amplitude and length of 1 and 60 m, respectively. The hull has two longitudinal bulkheads separated 2 m between them and double bottom of 1 m height, and is formed with 6 mm plating at the bottom, and 5 mm on the other parts, after subtracting the corrosion allowance; to support deck load there are two longitudinal girders at midpoint of span. Transverse frames are separated 55 cm. In a simplified way, no other longitudinal reinforcements need to be considered. (40)



2.- In a primary estimation of ship from problem 1, the shear stress distribution on the section shows a null value at a point located 2.0 m on each side from center line of the deck, calculate the maximum shear force that may be applied on the hull section. Consider an allowable shear stress of 105 N/mm^2 . (25)

3.- You have to complete the preliminary design of a watertight bulkhead of a ship with the following main dimensions: \triangle : 2700 tons, and, *L*: 80, *B*: 12, *D*: 6 and *T*: 4 meters. The wbkh has one transversal girder and several vertical stiffeners. The design process has already selected the thickness of the plating, 6.0 mm (with no corrosion allowance), and you have to select the number and characteristics of the stiffeners. To simplify the calculations take the stiffeners as flat bars of the same thickness as the plate of the bulkhead. If you need to combine stresses, consider a maximum

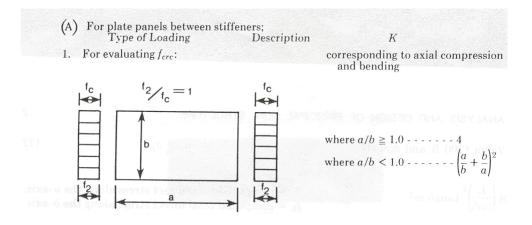
value of 200 N/mm² for the equivalent von Mises stress: $\sigma_{eq} = \sqrt{\sigma_1^2 - \sigma_1 \sigma_2 + \sigma_2^2}$. (35)

Useful formulations:

Buckling critical stress for plates in compression (DNV): $f_{crc}=f_{t.}$

$$f_t = f_e, \quad \frac{f_e}{f_y} \le 0.75, \qquad f_t = f_y \left(1 - \frac{3 f_y}{16 f_e} \right), \quad \frac{f_e}{f_y} > 0.75,$$

where the reference stress is: $f_e = 1.88E6 \left(\frac{t}{b}\right)^2 K$, kg/cm^2 , t and b in mm



Bending of isotropic rectangular plates, (Timoshenko):

TABLE	35.	Deflections	AND H	ENDING	MOMENT	rs in A	UNIF	ORMLY	LOADED
		RECTANGULAR	PLATE	WITH	BUILT-IN	EDGES	(FIG.	91)	
				$\nu =$	0.3				

·	b/a	$(w)_{x=0,y=0}$	$(M_x)_{x=a/2,y=0}$	$(M_y)_{x=0,y=b/2}$	$(M_x)_{x=0,y=0}$	$(M_y)_{x=0,y=0}$
UNC.	1.0	$0.00126ga^4/D$	$-0.0513qa^{2}$	$-0.0513qa^{2}$	0.0231qa2	0.0231qa2
	1.1	0.00150ga4/D	$-0.0581qa^{2}$	$-0.0538qa^{2}$	0.0264qa ²	0.0231qa2
170.	1.2	0.00172qa4/D	$-0.0639ga^2$	$-0.0554qa^{2}$	0.0299ga2	0.0228qa2
annak.	1.3	0.00191qa4/D	$-0.0687qa^{2}$	-0.0563qa ²	0.0327qa ²	$0.0222qa^2$
<u>9</u> 92→	1.4	$0.00207 qa^4/D$	-0.0726qa2	$-0.0568qa^{2}$	0.0349qa ²	0.0212qa2
	1.5	$0.00220 qa^4/D$	$-0.0757qa^{2}$	-0.0570qa2	0.0368qa ²	$0.0203qa^2$
	1.6	$0.00230 ga^4/D$	$-0.0780qa^{2}$	$-0.0571qa^{2}$	$0.0381qa^{2}$	0.0193qa ²
	1.7	$0.00238qa^4/D$	$-0.0799qa^2$	$-0.0571qa^{2}$	$0.0392 qa^2$	0.0182qa2
	1.8	0.00245qa4/D	$-0.0812ga^{2}$	$-0.0571qa^{2}$	0.0401ga ²	0.0174ya2
	1.9	$0.00249qa^4/D$	$-0.0822ga^2$	$-0.0571qa^{2}$	0.0407ga2	0.0165qa2
	2.0	$0.00254ga^4/D$	$-0.0829qa^2$	$-0.0571qa^{2}$	0.0412qa ²	0.0158ga2
	2.0	$0.00254qa^4/D$ 0.00260qa ⁴ /D	$-0.0833qa^2$	$-0.0571qa^2$	0.0417ga2	$0.0125qa^2$

I certify that during this exam I have complied with the Code of Ethics of our university.

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