Faculty of Maritime Engineering and Marine Sciences

Ship Structures II

Second evaluation

Jan. 29th, 2020

Student:

1.- Primary bending of a steel box barge with the following dimensions is to be analyzed: L: 25 m, B: 6 m, and D: 1.5 m, when the vessel is suspended from its ends by a crane. Hull structure includes two longitudinal girders welded to the bottom and deck plate. Total light weight of the vessel is 50 tons. Calculate hull maximum deflection, assuming weight distribution is uniform. (30)



Bottom plate: 5 mmLong.Girders: I_c : 1000cm⁴, A: 24cm², d_m : 15cm (from web bottom)Side plate: 4 mmDeck plate: 4 mmTransverse frame spacing: 50 cmTransverse frames: L60x60x5 mm

2.- If the framing system of ship in problem 1 is transversal, with deck frames (L60x60x5 mm) separated 50 cm, is it possible that deck plates may buckle in the above described situation? (20)

3.- The structure of a ship, L: 80 m, B: 12 m, D: 6 m and T: 4 m is to be designed. It has transverse framing and simple bottom, with a distribution that includes 2 longitudinal bulkheads and 6.0 m of separation between transverse bulkheads. Any other assumption is to be explained.

- a.- Rationally select spacing for primary and secondary stiffeners, and prepare a scheme of the primary structure. (10)
- b.- Calculate thickness for bottom plating with a 120 N/mm² allowable stress. Use the following formulation to estimate pressure: 10T+0.12L, kN/m^2 . (10)
- c.- According to DNV secondary stiffeners may support a 160 N/mm² normal stress, what dimensions (scantlings) would you recommend for them? (15)
- d.- Evaluate the possibility of failure of the *bottom plate* considering a combination of stresses. Clearly show the point of analysis. Use a -100 N/mm² value for the primary normal stress. Equivalent von Mises stress is: $\sigma_{eq} = \sqrt{\sigma_x^2 + \sigma_y^2 \sigma_x \sigma_y + 3\tau^2}$. (15)

Useful formulations:

Buckling critical stress for plates in compression (Taggart, ed., SDC, 1980): fcrc=ft.

$$f_t = f_e, \quad \frac{f_e}{f_y} \le 0.75, \qquad f_t = f_y \left(1 - \frac{3f_y}{16f_e} \right), \quad \frac{f_e}{f_y} > 0.75,$$

where the reference stress is: $f_e = 1.88E6 \left(\frac{t}{b}\right)^2 K$, kg/cm^2 , t and b in mm



Bending of isotropic rectangular plates, (Timoshenko):

| TABLE | 35. | DEFLECTIONS | AND E | BENDING | MOMENT | rs IN A | UNIF | ORMLY | LOADED |
|-------|-------------|-------------|-------|---------|----------|---------|-------|-------|--------|
| | | RECTANGULAR | PLATE | WITH | BUILT-IN | EDGES | (FIG. | 91) | |
| | $\nu = 0.3$ | | | | | | | | |

| × · · · · · | b/a | $(w)_{x=0,y=0}$ | $(M_x)_{x=a/2,y=0}$ | $(M_y)_{x=0,y=b/2}$ | $(M_x)_{x=0,y=0}$ | $(M_y)_{x=0,y=0}$ | |
|--------------------------|-----|------------------|---------------------|------------------------|-----------------------|-----------------------|--|
| Sig | 1.0 | 0.00126qa4/D | -0.0513qa2 | -0.0513qa ² | 0.0231qa ² | 0.0231qa ² | |
| | 1.1 | $0.00150ga^4/D$ | $-0.0581qa^{2}$ | $-0.0538qa^{2}$ | $0.0264qa^2$ | 0.0231qa ² | |
| 100 | 1.2 | $0.00172ga^4/D$ | $-0.0639ga^2$ | $-0.0554qa^2$ | 0.0299qa ² | 0.0228qa ² | |
| anna k. | 1.3 | 0.00191qa4/D | $-0.0687qa^{2}$ | -0.0563qa ² | 0.0327qa2 | 0.0222qa2 | |
| k <u>0</u> →k <u>0</u> → | 1.4 | 0.00207ga4/D | $-0.0726qa^2$ | $-0.0568qa^{2}$ | 0.0349ga2 | 0.0212qa2 | |
| , | 1.5 | $0.00220 ga^4/D$ | $-0.0757ga^{2}$ | $-0.0570qa^2$ | 0.0368ga2 | 0.0203qa2 | |
| | 1.6 | 0.00230ga4/D | $-0.0780qa^{2}$ | $-0.0571qa^2$ | 0.0381qa ² | 0.0193qa ² | |
| | 1.7 | $0.00238qa^4/D$ | $-0.0799qa^2$ | $-0.0571qa^{2}$ | 0.0392ga2 | 0.0182qa ² | |
| | 1.8 | 0.00245aa4/D | $-0.0812ga^{2}$ | $-0.0571qa^{2}$ | 0.0401qa ² | 0.0174ya2 | |
| | 1.9 | $0.00249 ga^4/D$ | $-0.0822qa^{2}$ | $-0.0571qa^2$ | 0.0407qa ² | 0.0165qa ² | |
| | 2.0 | $0.00254ga^4/D$ | $-0.0829qa^{2}$ | $-0.0571qa^{2}$ | 0.0412qa ² | 0.0158ga2 | |
| | 00 | $0.00260 ga^4/D$ | $-0.0833qa^{2}$ | $-0.0571qa^{2}$ | 0.0417qa2 | $0.0125qa^2$ | |

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I certify that during this exam I have complied with the Code of Ethics of our university.

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